Energy Harvesting Under Uncertainty

S Adhikari

1College of Engineering, Swansea University, Singleton Park, Swansea SA2 8PP, UK

IIT Madras, India
Collaborations

- Professor Mike Friswell and Dr Alexander Potrykus (Swansea University, UK).
- Professor Dan Inman (University of Michigan, USA)
- Professor Grzegorz Litak (University of Lublin, Poland).
- Professor Eric Jacquelin (University of Lyon, France)
- Professor S Narayanan and Dr S F Ali (IIT Madras, India).

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Piezoelectric vibration energy harvesting

The harvesting of ambient vibration energy for use in powering low energy electronic devices has formed the focus of much recent research.

Of the published results that focus on the piezoelectric effect as the transduction method, almost all have focused on harvesting using cantilever beams and on single frequency ambient energy, i.e., resonance based energy harvesting. Several authors have proposed methods to optimize the parameters of the system to maximize the harvested energy.

Some authors have considered energy harvesting under wide band excitation.
Why uncertainty is important for energy harvesting?

- In the context of energy harvesting of ambient vibration, the input excitation may not be always known exactly.
- There may be uncertainties associated with the numerical values considered for various parameters of the harvester. This might arise, for example, due to the difference between the true values and the assumed values.
- If there are several nominally identical energy harvesters to be manufactured, there may be genuine parametric variability within the ensemble.
- Any deviations from the assumed excitation may result an optimally designed harvester to become sub-optimal.
Types of uncertainty

Suppose the set of coupled equations for energy harvesting:

\[ \mathcal{L}\{u(t)\} = f(t) \]  

(1)

Uncertainty in the input excitations

- For this case in general \( f(t) \) is a random function of time. Such functions are called random processes.
- \( f(t) \) can be stationary or non-stationary random processes

Uncertainty in the system

- The operator \( \mathcal{L}\{\cdot\} \) is in general a function of parameters \( \theta_1, \theta_2, \ldots, \theta_n \in \mathbb{R} \).
- The uncertainty in the system can be characterised by the joint probability density function \( p_{\Theta_1, \Theta_2, \ldots, \Theta_n} (\theta_1, \theta_2, \ldots, \theta_n) \).
SDOF electromechanical models

Schematic diagrams of piezoelectric energy harvesters with two different harvesting circuits. (a) Harvesting circuit without an inductor, (b) Harvesting circuit with an inductor.
Governing equations

For the harvesting circuit **without** an inductor, the coupled electromechanical behavior can be expressed by the linear ordinary differential equations

\[ m\ddot{x}(t) + c\dot{x}(t) + kx(t) - \theta v(t) = f(t) \]  \hspace{1cm} (2)

\[ \theta \dot{x}(t) + C_p \dot{v}(t) + \frac{1}{R_l} v(t) = 0 \]  \hspace{1cm} (3)

For the harvesting circuit **with** an inductor, the electrical equation becomes

\[ \theta \ddot{x}(t) + C_p \ddot{v}(t) + \frac{1}{R_l} \dot{v}(t) + \frac{1}{L} v(t) = 0 \]  \hspace{1cm} (4)
Simplified piezomagnetoelastic model

Schematic of the piezomagnetoelastic device. The beam system is also referred to as the ‘Moon Beam’.
The nondimensional equations of motion for this system are

\[ \ddot{x} + 2\zeta\dot{x} - \frac{1}{2}x(1 - x^2) - \chi v = f(t), \quad (5) \]

\[ \dot{v} + \lambda v + \kappa\dot{x} = 0, \quad (6) \]

where \( x \) is the dimensionless transverse displacement of the beam tip, \( v \) is the dimensionless voltage across the load resistor, \( \chi \) is the dimensionless piezoelectric coupling term in the mechanical equation, \( \kappa \) is the dimensionless piezoelectric coupling term in the electrical equation, \( \lambda \propto 1/R_lC_p \) is the reciprocal of the dimensionless time constant of the electrical circuit, \( R_l \) is the load resistance, and \( C_p \) is the capacitance of the piezoelectric material. The force \( f(t) \) is proportional to the base acceleration on the device. If we consider the inductor, then the second equation will be \( \ddot{v} + \lambda \dot{v} + \beta v + \kappa\dot{x} = 0 \).
Possible physically realistic cases

Depending on the system and the excitation, several cases are possible:

- Linear system excited by harmonic excitation
- Linear system excited by stochastic excitation
- Linear stochastic system excited by harmonic/stochastic excitation
- Nonlinear system excited by harmonic excitation
- Nonlinear system excited by stochastic excitation
- Nonlinear stochastic system excited by harmonic/stochastic excitation

This talk is focused on application of random vibration theory to various energy harvesting problems
Circuit without an inductor

Our equations:

\[ m\ddot{x}(t) + c\dot{x}(t) + kx(t) - \theta v(t) = -m\ddot{x}_b(t) \]  (7)

\[ \theta \dot{x}(t) + C_p \dot{v}(t) + \frac{1}{R_l} v(t) = 0 \]  (8)

Transforming both the equations into the frequency domain and dividing the first equation by \( m \) and the second equation by \( C_p \) we obtain

\[ \left( -\omega^2 + 2i\omega \zeta \omega_n + \omega_n^2 \right) X(\omega) - \frac{\theta}{m} V(\omega) = \omega^2 X_b(\omega) \]  (9)

\[ i\omega \frac{\theta}{C_p} X(\omega) + \left( i\omega + \frac{1}{C_p R_l} \right) V(\omega) = 0 \]  (10)
Circuit without an inductor

The natural frequency of the harvester, $\omega_n$, and the damping factor, $\zeta$, are defined as

$$\omega_n = \sqrt{\frac{k}{m}} \quad \text{and} \quad \zeta = \frac{c}{2m\omega_n}. \quad (11)$$

Dividing the preceding equations by $\omega_n$ and writing in matrix form one has

$$\begin{bmatrix} (1 - \Omega^2) + 2i\Omega\zeta & -\frac{\theta}{k} \\ i\Omega\frac{\alpha\theta}{C_p} & (i\Omega\alpha + 1) \end{bmatrix} \begin{Bmatrix} X \\ Y \end{Bmatrix} = \begin{Bmatrix} \Omega^2 X_b \\ 0 \end{Bmatrix}, \quad (12)$$

where the dimensionless frequency and dimensionless time constant are defined as

$$\Omega = \frac{\omega}{\omega_n} \quad \text{and} \quad \alpha = \omega_n C_p R_l. \quad (13)$$

$\alpha$ is the time constant of the first order electrical system, non-dimensionalized using the natural frequency of the mechanical system.
Inverting the coefficient matrix, the displacement and voltage in the frequency domain can be obtained as

\[
\begin{bmatrix}
X \\
V
\end{bmatrix} = \frac{1}{\Delta_1} \begin{bmatrix}
(i\Omega\alpha+1) & \frac{\theta}{k} \\
-i\Omega \frac{\alpha \theta}{C_p} & (1-\Omega^2) + 2i\Omega \zeta
\end{bmatrix} \begin{bmatrix}
\Omega^2 X_b \\
0
\end{bmatrix} = \begin{bmatrix}
(i\Omega\alpha+1)\Omega^2 X_b/\Delta_1 \\
-i\Omega^3 \frac{\alpha \theta}{C_p} X_b/\Delta_1
\end{bmatrix},
\]

(14)

where the determinant of the coefficient matrix is

\[
\Delta_1(i\Omega) = (i\Omega)^3 \alpha + (2 \zeta \alpha + 1)(i\Omega)^2 + \left(\alpha + \kappa^2 \alpha + 2 \zeta\right)(i\Omega) + 1
\]

(15)

and the non-dimensional electromechanical coupling coefficient is

\[
\kappa^2 = \frac{\theta^2}{kC_p}.
\]

(16)
Mean power

The average harvested power due to the white-noise base acceleration with a circuit without an inductor can be obtained as

\[
E\left[\tilde{P}\right] = E \left[\frac{|V|^2}{(R_l \omega^4 \Phi_{xb} \Phi_{xb})}\right] = \frac{\pi m \alpha \kappa^2}{(2 \zeta \alpha^2 + \alpha) \kappa^2 + 4 \zeta^2 \alpha + (2 \alpha^2 + 2) \zeta}.
\]

- From Equation (14) we obtain the voltage in the frequency domain as

\[
V = \frac{-i\Omega^3 \alpha \theta}{\Delta_1 (i\Omega)} \Phi_{xb}. \quad (17)
\]

- We are interested in the mean of the normalized harvested power when the base acceleration is Gaussian white noise, that is

\[
|V|^2/(R_l \omega^4 \Phi_{xb} \Phi_{xb}).
\]
Circuit without an inductor

The spectral density of the acceleration $\omega^4 \Phi_{xb}x_b$ and is assumed to be constant. After some algebra, from Equation (17), the normalized power is

$$\tilde{P} = \frac{|V|^2}{(R_\omega \omega^4 \Phi_{xb}x_b)} = \frac{k \alpha \kappa^2}{\omega^3} \frac{\Omega^2}{\Delta_1(i\Omega) \Delta^*_1(i\Omega)}. \quad (18)$$

Using linear stationary random vibration theory, the average normalized power can be obtained as

$$E[\tilde{P}] = E\left[\frac{|V|^2}{(R_\omega \omega^4 \Phi_{xb}x_b)}\right] = \frac{k \alpha \kappa^2}{\omega^3} \int_{-\infty}^{\infty} \frac{\Omega^2}{\Delta_1(i\Omega) \Delta^*_1(i\Omega)} \, d\omega \quad (19)$$

From Equation (15) observe that $\Delta_1(i\Omega)$ is a third order polynomial in $(i\Omega)$. Noting that $d\omega = \omega_n d\Omega$ and from Equation (15), the average harvested power can be obtained from Equation (19) as

$$E[\tilde{P}] = E\left[\frac{|V|^2}{(R_\omega \omega^4 \Phi_{xb}x_b)}\right] = m \alpha \kappa^2 I^{(1)} \quad (20)$$
Circuit without an inductor

\[ I^{(1)} = \int_{-\infty}^{\infty} \frac{\Omega^2}{\Delta_1(i\Omega)\Delta_1^*(i\Omega)} \, d\Omega. \] (21)

After some algebra, this integral can be evaluated as

\[
\begin{vmatrix}
0 & 1 & 0 \\
-\alpha & \alpha + \kappa^2\alpha + 2\zeta & 0 \\
0 & -2\zeta\alpha - 1 & 1 \\
\end{vmatrix}
\]

\[
\begin{vmatrix}
2\zeta\alpha + 1 & -1 & 0 \\
-\alpha & \alpha + \kappa^2\alpha + 2\zeta & 0 \\
0 & -2\zeta\alpha - 1 & 1 \\
\end{vmatrix}
\]

Combining this with Equation (20) we obtain the average harvested power due to white-noise base acceleration.
The normalized mean power of a harvester without an inductor as a function of $\alpha$ and $\zeta$, with $\kappa = 0.6$. Maximizing the average power with respect to $\alpha$ gives the condition $\alpha^2 (1 + \kappa^2) = 1$ or in terms of physical quantities $R_i^2 C_p (kC_p + \theta^2) = m$. 
Circuit with an inductor

The electrical equation becomes

\[
\theta \ddot{x}(t) + C_p \dot{v}(t) + \frac{1}{R_l} \dot{v}(t) + \frac{1}{L} v(t) = 0 \tag{23}
\]

where \( L \) is the inductance of the circuit. Transforming equation (23) into the frequency domain and dividing by \( C_p \omega_n^2 \) one has

\[
-\Omega^2 \frac{\theta}{C_p} X + \left(-\Omega^2 + i\Omega \frac{1}{\alpha} + \frac{1}{\beta}\right) V = 0 \tag{24}
\]

where the second dimensionless constant is defined as

\[
\beta = \omega_n^2 L C_p, \tag{25}
\]

Two equations can be written in a matrix form as

\[
\begin{bmatrix}
(1-\Omega^2) + 2i\Omega \zeta & -\frac{\theta}{k} \\
-\Omega^2 \frac{\alpha \beta \theta}{C_p} & \alpha (1-\beta \Omega^2) + i\Omega \beta
\end{bmatrix}
\begin{bmatrix}
X \\
V
\end{bmatrix}
= \begin{bmatrix}
\Omega^2 X_b \\
0
\end{bmatrix}. \tag{26}
\]
Inverting the coefficient matrix, the displacement and voltage in the frequency domain can be obtained as

\[
\begin{align*}
\begin{bmatrix} X \\ V \end{bmatrix} &= \frac{1}{\Delta_2} \begin{bmatrix} \alpha (1 - \beta \Omega^2) + i \Omega \beta & \frac{\theta}{k} \\ \Omega^2 \frac{\alpha \beta \theta}{C_p} & (1 - \Omega^2) + 2i \Omega \zeta \end{bmatrix} \begin{bmatrix} \Omega^2 X_b \\ 0 \end{bmatrix} \\
&= \begin{bmatrix} (\alpha (1 - \beta \Omega^2) + i \Omega \beta) \Omega^2 X_b / \Delta_2 \\ \Omega^4 \frac{\alpha \beta \theta}{C_p} X_b / \Delta_2 \end{bmatrix}
\end{align*}
\]

(27)

where the determinant of the coefficient matrix is

\[
\Delta_2(i\Omega) = (i\Omega)^4 \beta \alpha + (2 \zeta \beta \alpha + \beta) (i\Omega)^3 \\
+ \left( \beta \alpha + \alpha + 2 \zeta \beta + \kappa^2 \beta \alpha \right) (i\Omega)^2 + (\beta + 2 \zeta \alpha) (i\Omega) + \alpha.
\]

(28)
Circuit with an inductor

Mean power

The average harvested power due to the white-noise base acceleration with a circuit with an inductor can be obtained as

\[
E\left[\tilde{P}\right] = \frac{m_\alpha \beta \kappa^2 \pi (\beta + 2 \alpha \zeta)}{\beta (\beta + 2 \alpha \zeta)(1 + 2 \alpha \zeta)(\alpha \kappa^2 + 2 \zeta) + 2 \alpha^2 \zeta (\beta - 1)^2}.
\]

This can be obtained in a very similar to the previous case.
The normalized mean power of a harvester with an inductor as a function of $\alpha$ and $\beta$, with $\zeta = 0.1$ and $\kappa = 0.6$. 

Normalized mean power: numerical illustration
Optimal parameter selection

The normalized mean power of a harvester with an inductor as a function of $\beta$ for $\alpha = 0.6$, $\zeta = 0.1$ and $\kappa = 0.6$. The * corresponds to the optimal value of $\beta = 1$ for the maximum mean harvested power.
The normalized mean power of a harvester with an inductor as a function of $\alpha$ for $\beta = 1$, $\zeta = 0.1$ and $\kappa = 0.6$. The * corresponds to the optimal value of $\alpha (= 1.667)$ for the maximum mean harvested power.
Stochastic system parameters

- Energy harvesting devices are expected to be produced in bulk quantities
- It is expected to have some parametric variability across the ‘samples’
- How can we take this into account and optimally design the parameters?

The natural frequency of the harvester, $\omega_n$, and the damping factor, $\zeta_n$, are assumed to be random in nature and are defined as

$$\omega_n = \bar{\omega}_n \psi_\omega$$
$$\zeta = \bar{\zeta} \psi_\zeta$$

where $\psi_\omega$ and $\psi_\zeta$ are the random parts of the natural frequency and damping coefficient. $\bar{\omega}_n$ and $\bar{\zeta}$ are the mean values of the natural frequency and damping coefficient.
The average (mean) normalized power can be obtained as

\[
E[P] = E \left[ \frac{|V|^2}{(R/\omega^4 X^2_b)} \right] = \frac{\bar{k}\alpha k^2 \Omega^2}{\bar{\omega}_n^3} \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} \frac{f_{\psi_\omega}(x_1) f_{\psi_\zeta}(x_2)}{\Delta_1(i\Omega, x_1, x_2) \Delta_1^*(i\Omega, x_1, x_2)} \, dx_1 \, dx_2 \tag{31}
\]

where

\[
\Delta_1(i\Omega, \psi_\omega, \psi_\zeta) = (i\Omega)^3 \alpha + (2\bar{\zeta} \alpha \psi_\omega \psi_\zeta + 1) (i\Omega)^2 + \left( \alpha \psi_\omega^2 + \kappa^2 \alpha + 2\bar{\zeta} \psi_\omega \psi_\zeta \right) (i\Omega) + \psi_\omega^2 \tag{32}
\]

The probability density functions (pdf) of \(\psi_\omega\) and \(\psi_\zeta\) are denoted by \(f_{\psi_\omega}(x)\) and \(f_{\psi_\zeta}(x)\) respectively.
The mean power for various values of standard deviation in natural frequency with \( \bar{\omega}_n = 670.5 \text{ rad/s}, \psi_\zeta = 1, \alpha = 0.8649, \kappa^2 = 0.1185. \)
Stochastic System Parameters

The mean power

![Graph showing the mean harvested power versus standard deviation of natural frequency](image)

The mean harvested power for various values of standard deviation of the natural frequency, normalised by the deterministic power \( \bar{\omega}_n = 670.5 \text{ rad/s}, \psi_\zeta = 1, \alpha = 0.8649, \kappa^2 = 0.1185 \).
Optimal parameter selection

The optimal value of $\alpha$:

$$\alpha^2_{\text{opt}} \approx \frac{(c_1 + c_2 \sigma^2 + 3c_3 \sigma^4)}{(c_4 + c_5 \sigma^2 + 3c_6 \sigma^4)}$$  \hspace{1cm} (33)

where

$$c_1 = 1 + (4\bar{\zeta}^2 - 2) \Omega^2 + \Omega^4, \hspace{0.5cm} c_2 = 6 + \left(4\bar{\zeta}^2 - 2\right) \Omega^2, \hspace{0.5cm} c_3 = 1,$$  \hspace{1cm} (34)

$$c_4 = \left(1 + 2\kappa^2 + \kappa^4\right) \Omega^2 + \left(4\ddot{\zeta}^2 - 2 - 2\kappa^2\right) \Omega^4 + \Omega^6,$$  \hspace{1cm} (35)

$$c_5 = \left(2\kappa^2 + 6\right) \Omega^2 + \left(4\ddot{\zeta}^2 - 2\right) \Omega^4, \hspace{1.5cm} c_6 = \Omega^2,$$  \hspace{1cm} (36)

and $\sigma$ is the standard deviation in natural frequency.
Optimal parameter selection

The optimal value of $\kappa$:

$$\kappa_{opt}^2 \approx \frac{1}{(\alpha \Omega)} \sqrt{(d_1 + d_2 \sigma^2 + d_3 \sigma^4)} \quad (37)$$

where

$$d_1 = 1 + \left(4\zeta^2 + \alpha^2 - 2\right) \Omega^2 + \left(4\zeta^2 \alpha^2 - 2\alpha^2 + 1\right) \Omega^4 + \alpha^2 \Omega^6 \quad (38)$$

$$d_2 = 6 + \left(4\zeta^2 + 6\alpha^2 - 2\right) \Omega^2 + \left(4\zeta^2 \alpha^2 - 2\alpha^2\right) \Omega^4 \quad (39)$$

$$d_3 = 3 + 3\alpha^2 \Omega^2 \quad (40)$$
Nonlinear coupled equations

\[
\ddot{x} + 2\zeta \dot{x} + g(x) - \chi \nu = f(t) \tag{41}
\]
\[
\dot{\nu} + \lambda \nu + \kappa \dot{x} = 0, \tag{42}
\]

The nonlinear stiffness is represented as \(g(x) = -\frac{1}{2}(x - x^3)\).

Assuming a non-zero mean random excitation (i.e., \(f(t) = f_0(t) + m_f\)) and a non-zero mean system response (i.e., \(x(t) = x_0(t) + m_x\)), the following equivalent linear system is considered,

\[
\ddot{x}_0 + 2\zeta \dot{x}_0 + a_0 x_0 + b_0 - \chi \nu = f_0(t) + m_f \tag{43}
\]

where \(f_0(t)\) and \(x_0(t)\) are zero mean random processes. \(m_f\) and \(m_x\) are the mean of the original processes \(f(t)\) and \(x(t)\) respectively. \(a_0\) and \(b_0\) are the constants to be determined with \(b_0 = m_f\) and \(a_0\) represents the square of the natural frequency of the linearized system \(\omega_{eq}^2\).
Linearisation Approach

**Linearised equations**

We minimise the expectation of the error norm i.e., $(\mathbb{E} \left[ \epsilon^2 \right], \text{with } \epsilon = g(x) - a_0x_0 - b_0)$. To determine the constants $a_0$ and $b_0$ in terms of the statistics of the response $x$, we take partial derivatives of the error norm $w.r.t.$ $a_0$ and $b_0$ and equate them to zero individually.

\[
\frac{\partial}{\partial a_0} \mathbb{E} \left[ \epsilon^2 \right] = \mathbb{E} \left[ g(x)x_0 \right] - a_0 \mathbb{E} \left[ x_0^2 \right] - b_0 \mathbb{E} \left[ x_0 \right] \quad (44)
\]

\[
\frac{\partial}{\partial b_0} \mathbb{E} \left[ \epsilon^2 \right] = \mathbb{E} \left[ g(x) \right] - a_0 \mathbb{E} \left[ x_0 \right] - b_0 \quad (45)
\]

Equating (44) and (45) to zero, we get,

\[
a_0 = \frac{\mathbb{E} \left[ g(x)x_0 \right]}{\mathbb{E} \left[ x_0^2 \right]} = \frac{\mathbb{E} \left[ g(x)x_0 \right]}{\sigma_x^2} \quad (46)
\]

\[
b_0 = \mathbb{E} \left[ g(x) \right] = m_f \quad (47)
\]
Simulated responses of the piezomagnetoelastic oscillator in terms of the standard deviations of displacement and voltage ($\sigma_x$ and $\sigma_v$) as the standard deviation of the random excitation $\sigma_f$ varies. (a) gives the ratio of the displacement and excitation; (b) gives the ratio of the voltage and excitation; and (c) shows the variance of the voltage, which is proportional to the mean power.
Phase portraits for $\lambda = 0.05$, and the stochastic force for (a) $\sigma_f = 0.025$, (b) $\sigma_f = 0.045$, (c) $\sigma_f = 0.065$. Note that the increasing noise level overcomes the potential barrier resulting in a significant increase in the displacement $x$. 
Voltage output due to Gaussian white noise ($\zeta = 0.01$, $\chi = 0.05$, and $\kappa = 0.5$ and $\lambda = 0.01$).
Voltage output due to Lévy noise ($\zeta = 0.01$, $\chi = 0.05$, and $\kappa = 0.5$ and $\lambda = 0.01$).
Inverted beam harvester

(a) Schematic diagram of inverted beam harvester, (b) a typical phase portrait of the tip mass.
Energy harvesting from bridge vibration

(a) Schematic diagram of a beam with a moving point load, (b) The variation in the energy generated by the harvester located at $L/3$ with $\alpha$ for a single vehicle traveling at different speeds, $u$
(a) Schematic diagram of energy harvesting dynamic vibration absorber attached to a single degree of freedom vibrating system, (b) Harvested power in $mW/m^2$ for nondimensional coupling coefficient $\kappa^2 = 0.3$
Summary of the results

- Vibration energy based piezoelectric and magnetopiezoelectric energy harvesters are expected to operate under a wide range of ambient environments. This talk considers energy harvesting of such systems under harmonic and random excitations.

- Optimal design parameters were obtained using the theory of linear random vibration.

- Nonlinearity of the system can be exploited to scavenge more energy over wider operating conditions.

- Uncertainty in the system parameters can have dramatic affect on energy harvesting. This should be taken into account for optimal design.

- Stochastic jump process models can be used for the calculation of harvested power.
Further details


Under Review
